

Cryogenic Heat Sink for Helium Gas Cooled Superconducting Power Devices

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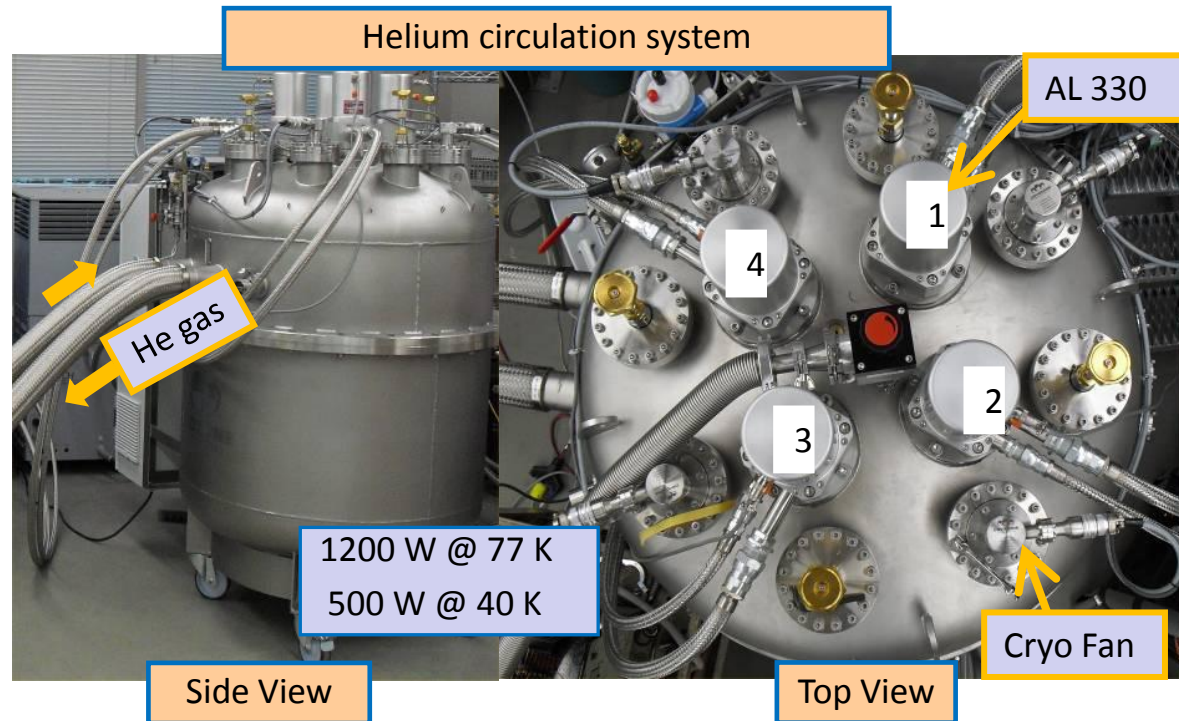
Excerpt from the Proceedings of the 2012 COMSOL Conference in Boston



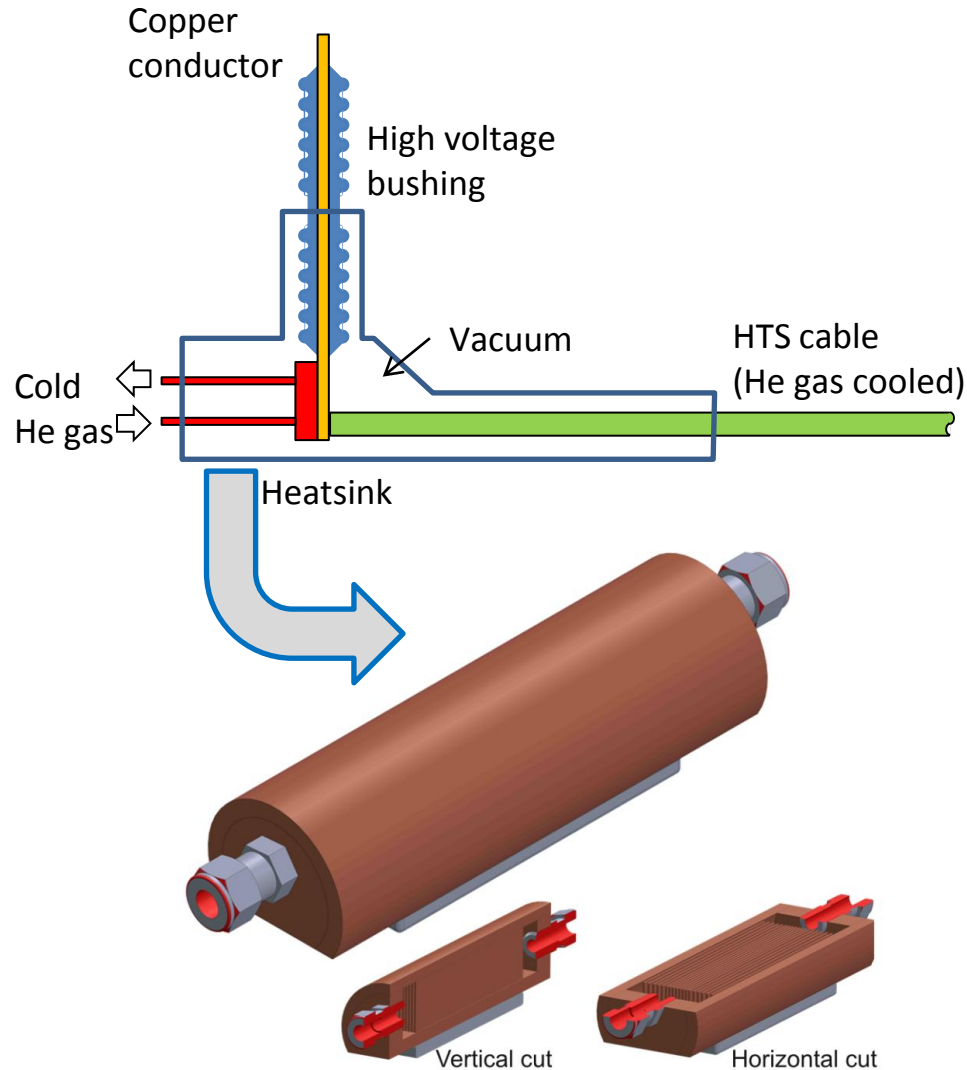
- Introduction
 - Superconducting power cables cooled by helium gas
 - Challenges in cable termination
 - Application of the heat sink
- Finite element model
 - 2D heat transfer
 - 3D fluid flow
- Experiment for model validation
- Conclusions

Superconducting Power Cables

- Superconducting cables for shipboard power system
 - Temperatures below LN2 → higher current density
 - Liquid cryogenes not permitted (asphyxiation & explosion hazard)
 - Solution: Helium gas at 50...60 K and 1.8 MPa; flow rate up to 20 g/s
- gHe has lower heat capacity than LN2
 - Cooling more challenging, especially at terminations

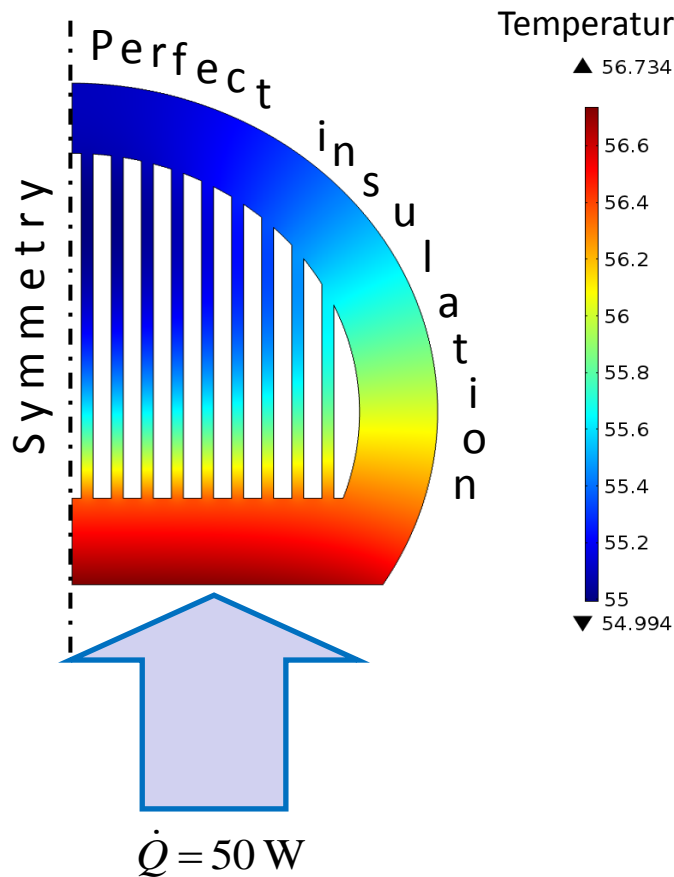


Model Heat Sink



- Problem:
 - Heat influx from ambient
 - Joule heating in bushing
- Solution:
 - Heat intercept attached to copper conductor
 - Cold He gas flow through heat sink
- Design:
 - Finned heat sink inside tube
 - Entirely made from copper
 - Design and dimensions need to be optimized by FEA
 - Small-scale model built for model validation

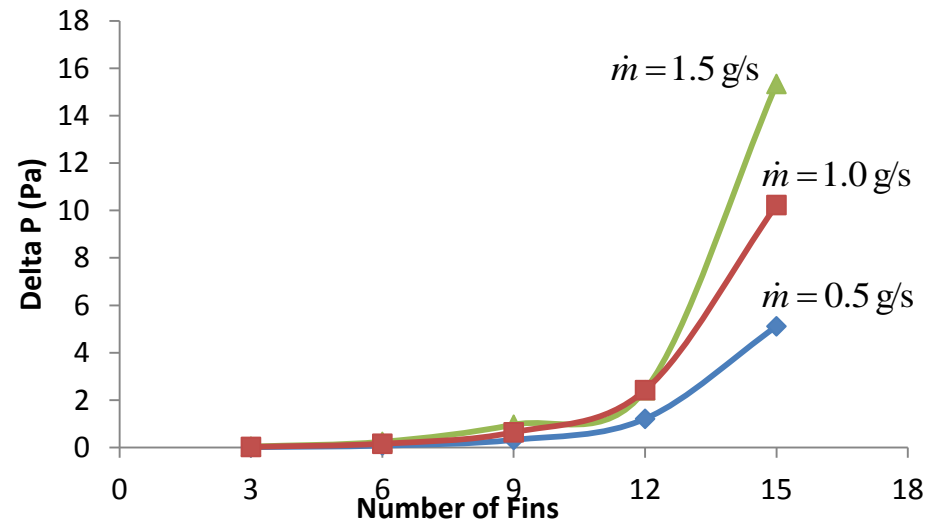
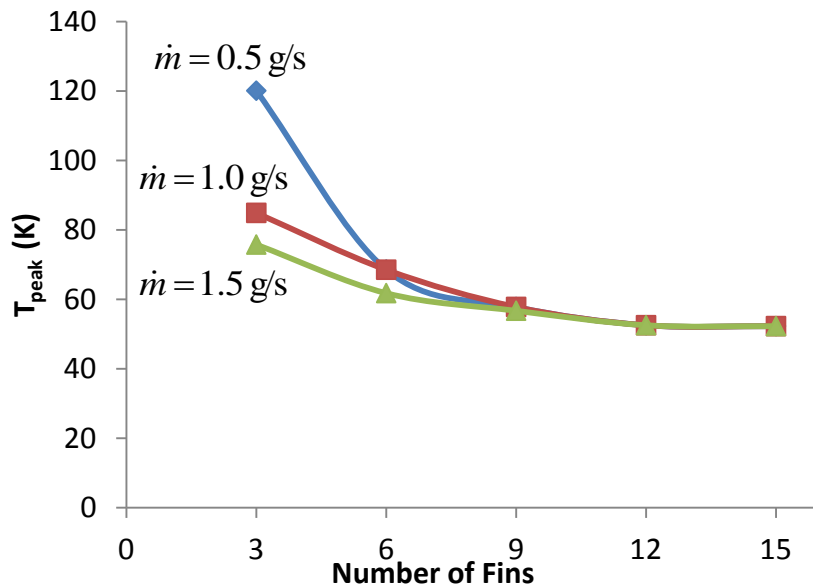
Goal: Determine optimal number of fins
Symmetry: All BC for $\frac{1}{2}$ heat sink



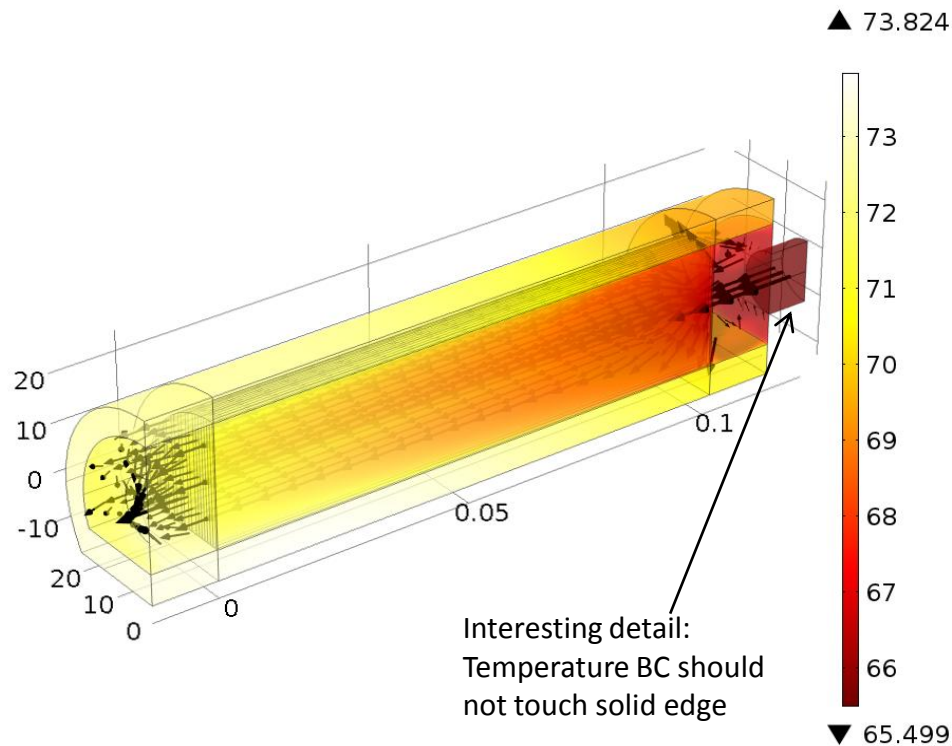
- Physics
 - *Heat Transfer in Solids*
 - No CFD, but
- Boundary conditions
 - Heat influx 50 W
 - Symmetry
 - Insulation (vacuum)
 - In channels: Convective cooling boundary condition ($h = 90 \text{ W/m}^2\text{K}$ for the 9-fin model, obtained by Dittus-Boelter correlation)
- Initial temperature: 50 K
- Material properties
 - Copper: k , c_p , ρ as a function of temperature
- Mesh size: normal (2986 elements for 9-fin model)
- Pressure drop calculated separately using Moody Diagram

Results from 2D Model

- Maximum heat sink temperature and pressure drop as a function of number of fins for three different mass flow rates
- 9 Fins seem to be optimal
- Flow rates of below 1 g/s are sufficient (50 W input power)



Finite Element Model: 3D Fluid Flow



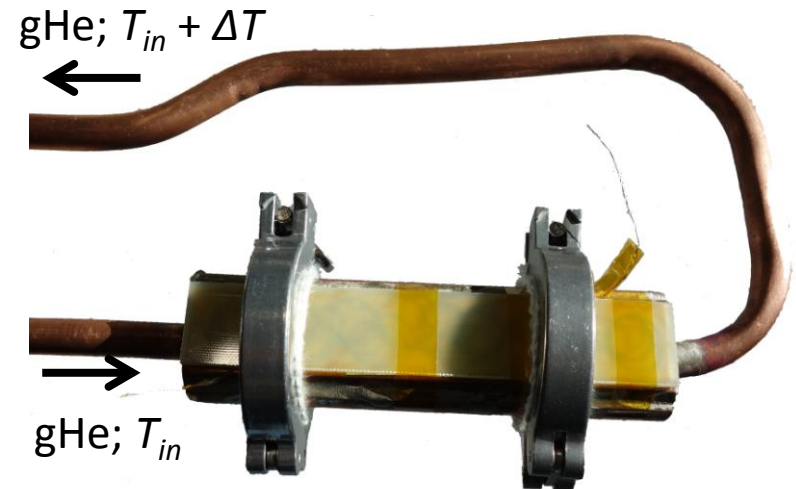
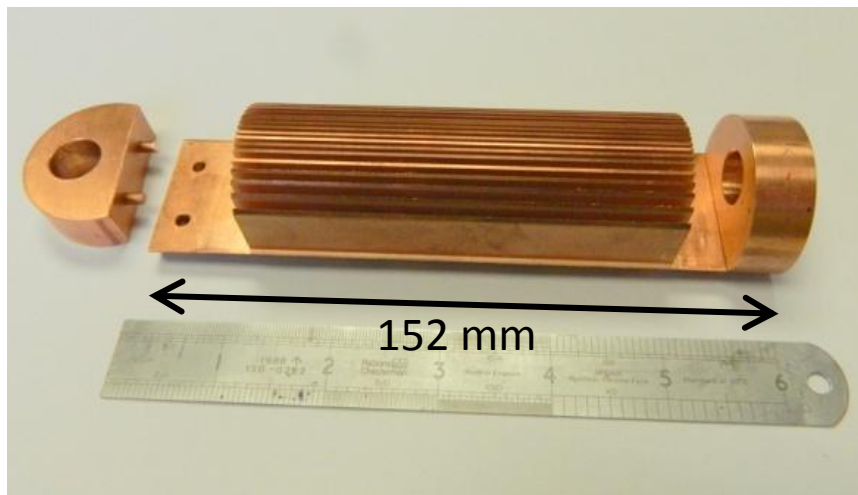
Temperature field for 100 W heat influx

Simulation time: 156 min (on 2x Intel Xeon X5570)

- Physics
 - *Conjugate Heat Transfer*
 - Laminar flow
- Boundary conditions
 - Inlet: Helium, 50 K, volume flow rate
 - Outlet: Pressure
 - Heat influx: same as 2D model
- Helium properties
 - Density considered constant (at 58 K)
- Mesh
 - Fluid domain: normal
 - Solid domain: finer
 - Total 1.54 million elements
- Convergence
 - GMRES: 240 iterations
 - Non-lin: 45 iterations

Experiment for Model Validation

- Made from copper
 - Most parts mechanically machined
 - Fins were machined by EDM
 - Joined by silver braze (optimal heat transfer; leak free)
- Heater based on resistance wire
- Wrapped in aluminized Mylar
- Installed in vacuum chamber



Parameter	50 W for full HS		100 W for full HS	
	Model	Experiment	Model	Experiment
Temperature inlet [K]	58.6	58.6	65.5	65.5
Temperature increase [K]	4.15	4.7	6.45	7.3
Temp. heat sink [K]	63.0	77.3	73.8	84.0
Pressure drop [Pa]	284	294	313	297

- Generally good agreement between simulation results and measurements
 - Except for heat sink temperature
 - Investigations under way to determine the reason for discrepancy (Model or measurement?)



- The chosen geometry is suitable
 - Low pressure drop
 - Excellent heat transfer
 - Higher flow rates for turbulent flow are under investigation
- The developed models are very useful tools for heat sink design and optimization
 - It will be used for a real application in near future
- Model will be extended to incorporate turbulent flow
- Optimization studies for geometrical parameters (non-uniform spacing of fins; thickness of fins)